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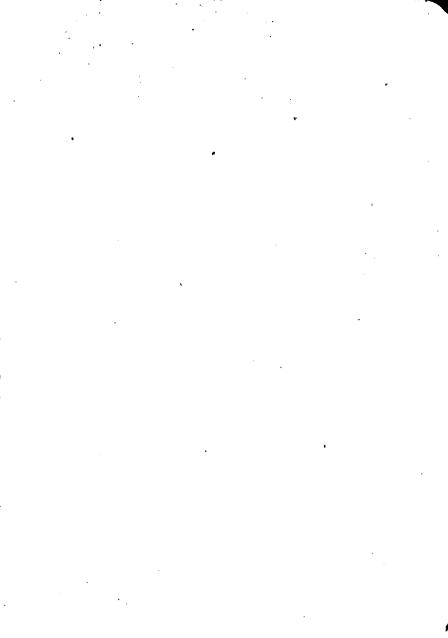
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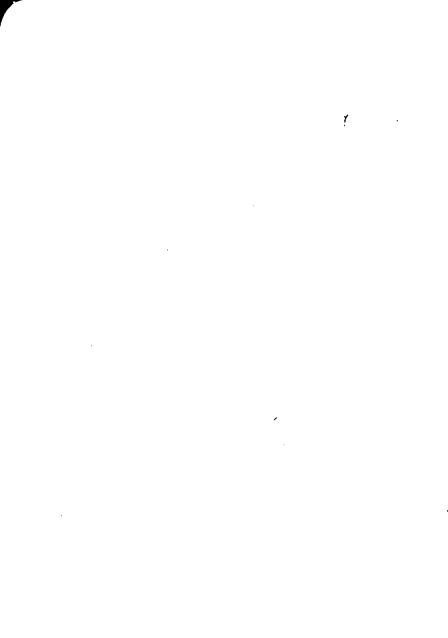
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# FIRE ENGINE TESTS

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# FIRE STREAM TABLES



NATIONAL BOARD OF FIRE UNDERWRITERS
'' New York

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> THE EVENING POST JOS PRINTING OFFICE 186 FULTON ST., N. Y.

### PREFACE.

This pamphlet has been prepared for the purpose of assisting fire department officials and others who may wish to determine the condition of fire engines. It may also be of service in testing the capacity of new engines with a view to their acceptance by a city.

Tests similar to those outlined herein have been adopted by several fire departments and are being made by our engineers in their investigation of cities throughout the country, so that by corresponding with this Board, the location of the nearest field party may be ascertained and if desired, an opportunity afforded to observe such tests.

The appended fire stream tables, on pages 26 to 47, are based on tests of rubber-lined fire hose made in October, 1909, by our engineers, with the assistance of the New York Fire Department and the co-operation of the Department of Water Supply of New York City. These tables may also be used to find the approximate amount of water used at a fire, if engineers will observe from time to time the water pressure carried and the length of time at work. With an approximate average of the water pressure at each engine, the amount of water delivered per minute can be found for each line if the size of nozzle and length of hose is also known. Copies of this pamphlet will be sent to such captains of companies and engineers of steamers as would use them in keeping accurate records of the performance of their engine at fires.

NATIONAL BOARD OF FIRE UNDERWRITERS

COMMITTEE ON FIRE PREVENTION,
135 William Street,
New York.

March, 1910.

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### PRACTICAL TESTS FOR FIRE ENGINES.

It is the purpose of this manual to set forth convenient and practical methods of making fire engine tests which will show the physical condition of engines, their capacity for delivering water at a reasonable pressure and the ability of the operating crews. The method described has been in use for a number of years and has been found practical, exact and of great value. Although methods similar to that described below are in use in some departments, the character of tests made in many cities, and especially those for acceptance, are usually more spectacular than exact. The throwing of a stream over a church spire, city hall or court house does not necessarily show that the engine is capable of delivering its full rated capacity at a proper working pressure.

Investigation has shown that where regular and systematic tests of engines are not made, even in well managed fire departments, defects often exist which may continue unsuspected for considerable periods and become manifest under the stress of a large fire, where the engine is called upon to deliver its full capacity under suitable working pressures. Such tests will bring to light numerous defects, as, for example, improper setting of steam valves, broken or worn pump valves, broken, weak or displaced valve springs, loose or tight bearings, worn or broken pump plungers, poor or defective condition of the boiler and poor quality of the coal supplied for engine fuel. Furthermore, regular tests are a most valuable drill for engine crews, for in only a few departments do they receive sufficient training in operating engines to capacity. The breakdown of an engine at a fire or the inability of the crew to operate it to capacity may be the direct cause of confusion and the needless loss of property and perhaps of life, to the discredit of the department.

Contracts for new fire engines usually contain guarantees that the engine will deliver a certain quantity of water, but often do not specify the pressure at which it is to be delivered, nor provide for any definite tests which will

accurately determine whether the engine has fulfilled the guarantee; or, in other words, if the department is getting what it is paying for. In several cities, engines are required to fill large measured tanks in a specified time, but this is a cumbersome method at best, and such tanks are frequently unavailable; this usually gives no definite results as to pressure obtained and power developed.

A practical test should show, with fair accuracy, the condition of both water and steam ends of pumps and the condition of the boiler; determine the amount of water which the engine will pump at a reasonable working pressure, such as would be required when operating at a large fire; demonstrate the ability of the engine to draft water, whether the pumps and waterways are tight under high pressures and steam valves are properly set, and whether the coal used is quick steaming and free from objectionable impurities. In addition, the test should be of such a character as to approach the working condition at a serious fire where the full capacity of the engine would be required, and at the same time be easily understood. The following tests bring out all of these points.

The displacement test indicates very closely the actual condition of the pumps as a whole and, in conjunction with the high pressure and valve tests, the condition of the plungers, pump valves, packing, etc. The high pressure test, in connection with the results obtained from the capacity test, indicates the setting of steam valves and condition of steam cylinders. The capacity test shows the steaming quality of the boiler under heavy draft and the ability of the engine to make sufficient speed to develop its capacity when working against a reasonable water pressure. test is made from a cistern or reservoir, it will show the ability of the engine to draft. If made from a hydrant, the percentage of slip obtained will indicate this feature as well. as an engine showing less than 7 per cent. slip may be depended upon to take suction satisfactorily. Incidentally, the test also shows the ability of the engine crew in operating and stoking the engine.

Any machine, when new, should be capable of greater work than after several years of service; for this reason, a new engine should be given an acceptance test at least as

 PARATUS FOR TESTING



severe as any work it may have to perform in actual service. This test should bring out not only the capacity to pump the actual volume of water specified by the maker as the rated capacity, but also to do this at a good working pressure. It is the opinion of many supervising engineers that this pressure should be at least 150 pounds if engine is likely to be required to draft, and as this does not seem too severe and is required in some specifications, the suggestion is made that engines purchased be required to have sufficient boiler capacity to give a net water pressure at acceptance test equivalent to the following values:

Hydrant Pressures Under Fire Draft. 50 lbs. or over.	Engine to Deliver Net Water Pressure of roo lbs.
30 " to 50 lbs.	120 "
10 " to 30 "	140 "
10 " or less.	150 "

Engines in service need not be given as severe a test as those being accepted, as it is mainly their general condition that is to be ascertained; for this reason, 100 pounds net water pressure would seem a sufficiently high requirement for the ordinary capacity test, which should be made at least yearly.

Apparatus Necessary for Testing.—For the tests outlined below, no elaborate or costly outfit is needed, the only special appliances absolutely required being as shown on Plate I and listed below:

A revolution counter. (Figure 3.)
A stop-watch. (Figure 5.)
A small Pitot tube. (Figure 8.)
Two or more pressure gages. (Figures 1 and 9.)
A set of smooth bore nozzles. (Figure 4.)
A hydrant or engine-discharge cap. (Figure 2.)

The revolution counter should be of a type easily attached to the engine frame, or any convenient part, and so made as to register accurately at any speed likely to be reached by a reciprocating engine and be easily read.

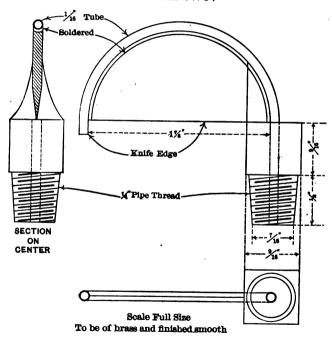
The counter may be provided with straps for attaching to engine, or with the clamp and angle iron shown on Plates I and II.

Tachometers and speed indicators are unsuitable for fire engine work, as the vibration is apt to render their readings unreliable.

A stop-watch can be purchased for less than \$10, although an ordinary watch can be used.

The Pitot tube may be any of several suitable types now on the market, or the type shown on Plate I may be readily constructed. Dimensions are given below. It should be connected by 1/4-inch brass pipe fittings to a pressure gage as shown.

### NOZZLE STREAM PITOT



The pressure gages should be preferably not more than 3½ inches in diameter, in order that they may be conveniently handled. They should be of the compound type, in order



# PLATE II. METHOD OF ATTACHING GAGES AND COUNTER FOR TESTING ENGINES.



that any disarrangement of the needle may be readily observed, one capable of indicating pressures from a vacuum up to 150 pounds and one up to 200 pounds, and preferably divided for every pound and marked every 5 or 10 pounds, as shown in Figures 1 and 9, Plate I. Gages, especially those used with the Pitot, should be of good quality and accurate. They should be carefully calibrated (tested) with a weight tester or a standard gage before each day's work.

Nozzles suitable for testing are usually found in the regular equipment of every fire department. Only smooth bore tapered nozzles should be used, as discharges from ring nozzles are uncertain. Care should be taken that the ting are not nicked or otherwise injured, and washers do not project into the pipe, as a perfectly smooth waterway is essential. The ring nozzles on many engines have loose rings, which may be slipped out by unscrewing the end cap, leaving a suitable smooth-bore tip. nozzles should not be used, as these generally have interior projections or breaks in the waterway, likely to cause eddies in the stream. Where much testing is to be done, it is better to set aside nozzles, keeping them solely for that purpose. The bore of nozzles should be accurate to size within 1/1,000 of an inch and carefully measured.

The engine-discharge cap, or hydrant cap (in most cities these have the same thread) is tapped for ¼-inch pipe thread and fitted with a nipple and stop-cock for attaching the test gage. By attaching to the discharge outlet of the engine as shown on Plate II, the engine water gage and the test gage may be compared to determine if the engine gage is correct. Where there is time to detach the water gage and a testing set is available, the gage can be more accurately checked. The steam gages are less likely to get out of order, being less subject to sudden fluctuations, and a comparison of readings of side and rear steam gages will usually be sufficient. If the engine has no suction gage or tapped suction cap, the engine or hydrant cap should be used on the second outlet of the hydrant when testing an engine at a double outlet hydrant.

Tests are best made by a supervisor (as the master mechanic or other officer conducting the test will hereafter

be called), with an assistant accustomed to reading gages. Tables showing the discharge at various pressures through different nozzles, for use with Pitot tube readings, are to be found on pages 24 and 25. A suitable form for recording data of tests is shown on page 14, and until the supervisor becomes familiar with tests, it is advisable to use a similar form at the tests in order not to overlook any necessary data. Later, a pocket note-book will doubtless be found more convenient, care being taken to record all the necessary data.

Preliminary to Test.—If possible, calibrate gages of engine before the test, by detaching and comparing on a portable gage-testing set. They should be calibrated in the position in which they are to be used, either horizontally or vertically. If this is not done, check water and suction gages at test, as explained below.

If it is desired to determine the ability of the regular engine crew, the engine should, of course, be operated by them; if the condition and capacity of the engine are the unknown factors, a crew known to be efficient should be selected.

If there is any convenient body of water, or cistern, where water may be drafted with not over 10 feet of lift, then test should be made at draft; otherwise, attach engine to hydrant. care being taken to get a hydrant attached to a large main (8-inch or larger), and that the hydrant pressure is not excessive, preferably below 40 pounds. Four-inch or larger suction should be used. After suitably stationing engine, light the fire; note the time when smoke comes from stack, when steam gage needle moves, at 50 pounds of steam, at 100 pounds, and pressure and time of blowing off. has hot water in boiler, this may be omitted, noting only the pressure at which safety valve blows off. Then, if water gage on engine has not been calibrated (checked), attach hydrant cap and 200-pound test gage to engine discharge outlet, as shown on Plate II. Record zero of all three gageswater, suction and test gages: open hydrant and record static pressure on all three gages; then with churn (hand relief) valve partly open and discharge gates shut, pump up pressure and compare test and water gages at 80 pounds, 100, 110, 120, etc., up to 110 pounds over the static or hydrant pressure. If engine has no suction gage, one of the suction caps on the engine can be tapped to connect the gage, as shown on Plate II, or the engine or hydrant cap provided with the second gage should be attached to one hydrant outlet.

Let supervisor and assistant compare watches and set second hands together, or nearly so; this is more quickly accomplished if one watch has a stop-hand. The supervisor will find it convenient to tie his watch to coat or wrist in order to leave his hands free to hold note-book or Pitot. A leather watch holder and wrist strap, as shown on Plate I, such as any harness maker can make, is a convenient appliance for this purpose. Attach the revolution counter and connect with one of the eccentric strap oil cups or studs by a short length of cord, as shown on Plate II; have engine started slowly and adjust counter cord so that each revolution registers.

Displacement and Capacity Test.—While the engine is getting up steam, have firemen lay hose and connect nozzle. If testing on a paved street, it is best to lay nozzle down in gutter. Use a play-pipe holder or tie nozzle to any convenient post, in order to prevent pipe getting away from pipeman and doing damage.

For the larger engines, attach a line of hose on each side of the engine and connect into the siamese of a deluge set.

With the smaller size engines, it is usually more convenient to use a single line from one side of the engine; when deluge sets are not available, single lines may be used on the larger engines. In the tables on pages 18 and 19, the length of hose and size of nozzle best adapted for testing engines of various sizes are given. In testing with the siamesed lines, start the engine with both lines open and bring it up to speed; if the desired water pressure is not obtained, close the discharge gate on one line slowly until the gage indicates the proper pressure. Similarly, with a single line attached, the gate is closed slowly after engine has obtained its full speed until the desired pressure is obtained.

The supervisor can, from time to time, regulate this discharge gate to keep the desired water pressure, although if the crew operates the engine properly but little change will have to be made throughout the test. The engineer

can be instructed to direct all his attention to operating his engine to full capacity, and the supervisor or testing engineer can regulate the water pressure, take the readings of the revolution counter, steam, water and suction gages, while his assistant takes readings of the nozzle pressure throughout the test.

When siamesed lines are used, should the engine not be able to maintain the desired water pressure with one line shut off entirely, add another length of hose to each side, or use a nozzle 1/2-inch smaller. With single lines, when the engine cannot maintain the desired pressure without undue throttling of the discharge valve, use a smaller nozzle or add another length of hose. The nozzle readings should, if possible, be over 40 pounds, as below this point readings must be very nearly constant to give accurate results.

Should water pressure at the engine be too high with both lines wide open, use a larger nozzle or cut out a length of hose from each side.

Relief valves should be closed, sprinkler used only as needed, and feed pumps operated regularly. The capacity test should last at least 20 minutes from the time the engine reaches full speed. During this time the water pressure at the engine should be constant and such as to give a net water pressure over the suction pressure as given on page 5. In all cases at least 100 pounds net pressure should be held. Unless the rubber tires cause undue vibration, a modern engine, if in good condition, can safely run for an indefinite period at 400 to 425 feet of piston travel per minute, that is, 300 to 320 revolutions for an 8-inch stroke.

It is usually better to hold about 10 pounds over the pressure actually required, when the water pressure fluctuates much, as most engineers read the top of swing of a gage needle, while the supervisor, of course, should read the middle of the vibration. Gages may be throttled to prevent excessive vibration, but should always show some vibration to get true readings. During the capacity test, the supervisor should read counter (exactly at minute) and steam, water and suction gages each minute in regular order, and note the handling and stoking, feed water, leaks, uneven steam pressure, blowing off, foaming of boiler, accidents, and the other little details

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which his experience teaches him to observe. Meanwhile the supervisor's assistant should read the nozzle pressure every ½ minute. Special care should be taken in reading the nozzle pressure. The Pitot should be held in the middle of the stream, with the tip about one-half the diameter of the bore from the end of the nozzle. Gage should be horizontal or vertical, according to the position in which it was calibrated, and at the same level as the end of the nozzle. This is shown on Plate III.

High Pressure Test.—After a run of 20 minutes in which there were no serious interruptions to readings, and pressure was maintained at an average of at least 100 pounds net, stop stoking; shut down, close discharge gates, partly open churn valve and get steam down to between 70 and 80 pounds, drawing fire if necessary. Then start engine slowly, and gradually close churn valve tight. See that all other openings, feed pumps, sprinklers, relief cocks, etc., are shut. Let engine turn in this condition for one or two minutes: observe the number of revolutions, and the water, steam and suction (now static) pressures: note any uneven motion of engine, blowing through of steam or imperfect valve setting, leaks in steam or water ends, or fittings, etc. If pumps are in good condition and valves set correctly, speed should not be over one revolution in 10 seconds in any modern type engine. (This does not apply to a Silsby or a Button.) With 70 pounds steam and 50 pounds suction, water pressure will reach about 250 pounds; this is perfectly safe and not a severe test, as such pressures are frequently met in operation when long lines are used.

Valve Tests.—After taking the observations for the high pressure test, shut off throttle of engine and open cylinder drips. Note the drop in water pressure for say one-half minute. The manner in which this pressure holds up is an indication of the condition of the discharge valves. A drop of not over 15 pounds in one-half minute, provided there are no external leaks visible around the pump, indicates a fairly good condition of the valves.

Suction Test.—If the engine has been tested at a hydrant, its ability to draft may be determined as follows, provided it is equipped with a compound suction gage or one of the suction caps is tapped to receive a compound gage: Discon-

nect engine from hydrant while there is still some steam pressure on boiler, put both suction caps on tight, open one of the discharge gates and then open throttle, allowing engine to run at a moderate speed, observe the reading of the compound gage while running, and also after shutting down. The drop of the vacuum after shutting down is an indication of the condition of the suction valves, provided all joints are good.

To Figure Displacement.—(Displacement is figured as indicated for sample test, pages 14 and 15.) In averaging the nozzle, steam, water and suction pressures, subtract ½ of first and last readings from sum of readings used (see page 15 and sample test sheet). Average the nozzle pressure during a period in which the engine ran steadily, water pressure was well maintained and the nozzle pressure varied the least. When possible, use a 20-minute period in figuring the displacement; if for any reason there is much variation in the nozzle pressure, say over 10 per cent. during any one minute, select as long a period as possible, but at least 10 minutes, during which the pressure has been well maintained. Correct for gage error. Take out corresponding gallons from table, pages 24 and 25, interpolating for odd pressures or for odd sized nozzles.

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and 1/16" "	rence in nozz " " " 9/16" nozzle	66	nds gives	45 gallons

Divide the average gallons discharged by the average revolutions per minute to obtain the actual net displacement of the pumps. The nominal displacement will be found from the table, page 16, allowing for the pump rods. The dimension of the pumps, such as stroke, diameter of pump barrel and pump rods, should be accurately measured, if in question. The difference between actual and nominal displacements is the slip, which should be from 3 to 5 per cent, of the nominal displacement in a new engine (6 per cent. in a rotary); of this, about 1/2 per cent, is due to the feed water (I per cent. with a Button or Silsby engine). After engine has been in use a few months, slip will generally increase about I per cent: thereafter, if valves and packings are given proper attention, there should be only a slight increase. A slip of 10 per cent, or over indicates broken or displaced valve springs. and more than this, a badly worn plunger or pump barrel, or possibly a leaky suction. In a rotary, the wear is principally in the pump cam slides, which will also stick at times, causing increased slip even if not worn.

To Figure Capacity.—When the engine is run for 20 minutes at a uniform speed during the displacement test, the average discharge measured at the nozzle by the Pitot is the capacity of the engine. If only a to-minute period of the run is used for figuring the displacement, the capacity of the engine is determined by multiplying the actual displacement (found in the displacement test) by the average revolutions per minute during a 20-minute period in which the engine worked at its full capacity. Steam, water and suction pressures during the capacity run should be averaged and corrected for gage error. In figuring percentage of capacity delivered, for a new fire engine, it is well to use contract figures for the rated capacity which the engine is guaranteed to deliver. A capacity due to a piston travel of about 420 feet per minute (315 revolutions for 8-inch. stroke) less a 3 per cent. allowance for slip, is reasonable for a modern engine; older types vary considerably.

### LOG OF FIRE ENGINE TEST

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Ave Corr Corr Gall Rev Disp	nor proceed per management of the per manage	ores 9 min 7	20 m 9.7 2 14.7 14.7 14.9 12.4 13.1/	-	Dura Ave. Gallo Ave.s Ave.s	tion m c prit as per steam sater p	min 2.0 324 min 732 pres 193 pres 194 res 354	1851 34.6		4.05 4.06	SESSO	RPM.	STEAM 72 72 72	WATER 220	SUCTION TIME	WATER 130 125  N TEST SUCTION GAR 20"
Ave Corr Corr Gall Rev Disp	nor proceed some per or per or laceme	ores 9 min 7	20 m 19.7 2 14.7 14.7 124.4 1311 1370	-	Dura Ave. Gallo Ave.s Ave.s	tion m c prit as per steam sater p	min 2.0 324 min 732 pres 193 pres 194 res 354	1851 34.6		4.05 4.06	SESSO	RPM.	STEAM 72 72 72	WATER 220	SUCTION TIME	WATER 130 125  N TEST SUCTION GAR 20"
Ave Corr Corr Gall Rev Disp	nor proceed some per or per or laceme	ores 9 min 7	20 m 19.7 2 14.7 14.7 124.4 1311 1370	-	Dura Ave. Gallo Ave.s Ave.s	tion m c prit as per steam sater p	min 2.0 324 min 732 pres 193 pres 194 res 354	1851 34.6		4.05 4.06	SESSO	RPM.	STEAM 72 72 72	WATER 220	SUCTION TIME	WATER 130 125  N TEST SUCTION GAI 20"
Ave Corr Corr Gall Rev Disp	nor proceed some per or per or laceme	ores 9 min 7	20 m 19.7 2 14.7 14.7 124.4 1311 1370	-	Dura Ave. Gallo Ave.s Ave.s	tion m c prit as per steam sater p	min 2.0 324 min 732 pres 193 pres 194 res 354	1851 34.6		4.05 4.06	522F 5230 SARKS	Z Z	STEAM 72 72 72	WATER 220 240	SUCTION TIME	WATER 130 125  N TEST SUCTION GAI 20"
Ave Corr Corr Gall Rev Disp	nor proceed some per or per or laceme	ores 9 min 7	20 m 19.7 2 14.7 14.7 124.4 1311 1370	-	Dura Ave. Gallo Ave.s Ave.s	tion m c prit as per steam sater p	min 2.0 324 min 732 pres 193 pres 194 res 354	1851 34.6		4.05 4.06	SEES SEES GREAT	nen.	STEAM 72 72 72 Rote	WATER 220 200	TIME 487 4882 SUCTION TIME 4.10	WATER 130 125  N TEST SUCTION GAI 20"
Ave Corr Corr Gall Rev Disp	nor proceed some per or per or laceme	ores 9 min 7	20 m 19.7 2 14.7 14.7 124.4 1311 1370	-	Dura Ave. Gallo Ave.s Ave.s	tion m c prit as per steam sater p	ins 2.6 325 min 756 more 153 mres 164 ces 354	1851 34.6		4.05 4.06	SEES SEES SEES SEES SEES SEES SEES SEE	nen.	STEAM 72 72 72 Gate Gate 700	WATER 220 200 200 4incit	TIME \$07 \$082 SUCTION TIME \$10 \$4.11	WATER 130 125  N TEST SUCTION GAI 20"
Ave Corr Corr Gall Rev Disp	nor proceed some per or per or laceme	ores 9 min 7	20 m 19.7 2 14.7 14.7 124.4 1311 1370	-	Dura Ave. Gallo Ave.s Ave.s	tion m c prit as per steam sater p	min 2.0 324 min 732 pres 193 pres 194 res 354	1851 34.6		4.05 4.06	SEES SEES SEES SEES SEES SEES SEES SEE	nen.	STEAM 72 72 72 Gate Gate 700	WATER 220 200 200 4incit	TIME \$07 \$082 SUCTION TIME \$10 \$4.11	WATER 130 125  N TEST SUCTION GAI 20"
Ave Corr Corr Gall Rev Disp	nor proceed some per or per or laceme	ores 9 min 7	20 m 19.7 2 14.7 14.7 124.4 1311 1370	-	Dura Ave. Gallo Ave.s Ave.s	tion m c prit as per steam sater p	ins 2.6 325 min 756 more 153 mres 164 ces 354	1851 34.6		4.05 4.06	SEES SEES SEES SEES SEES SEES SEES SEE	nen.	STEAM 72 72 72 Rote	WATER 220 200 200 4incit	TIME  487  4884  SUCTION  TIME  4:10  A.11	WATER 130 125  N TEST SUCTION BA 20°

### CALCULATIONS FOR ENGINE TESTS.

### (FOR TEST ON OPPOSITE PAGE.)

DISPLACEMENT TEST. AVERAGE DISCHARGE. To obtain Average Nozzle Pressure; Sum Column "Min."	CAPACITY TEST.  AVERAGE R. P. M.  Same as for Displacement Test in this case.
Subtract 1/2 sum of first and last figures	GALLONS PER MINUTE. Same as for Displacement Test in this case.
Sum Column "14" 1.791 " " 1,795 " " 1,802  Divide by 80 7.178	AVERAGES OF PRESSURES.  Steam: Sum of Column
Average Nozzle Reading. 89.7 Correction from Gage Test Sheet+2.0	Divide by 20
Average Nozzie Pressure. 91.7 From Discharge Tables for 156"	Water: Sum of Column
Nozzle: 92 lbs. gives	Divided by 20) 2,922.5
1.7 lbs. gives 6.8 " Then 91.7 lbs. = 749.8 gallons.	Average Reading 146.1 Correction from Test of Gage and Test Sheet, for Gage No. 119
AVERAGE R. P. M.	Average Water Pressure 145.1
Counter at 8.59	Suction: Sum of Column
Average R. P. M = 824.4	Divide by 90
ACTUAL DISPLACEMENT.  Average Discharge 749.8	Average Reading 85.6 Correction from Test of Gage $+1.0$
Average R. P. M. = 324.4 = 9.811	Average Suction Pressure 86,6
NOMINAL DISPLACEMENT. From Engine Displacement Table: 43" Bore, 8" Stroke 2,455	Not Pressure: Average water pressure 145.1 Average suction pressure
1½" Pump Rod	Average net pressure 108.5
Nominal Displacement = 2.870	PERCENTAGE OF CAPACITY OBTAINED. Reasonable capacity of
SLIP, IN PER CENT.  Nom. Displacem't — Act. Displacem't  Nominal Displacement  2.870 — 2.811  2.870 = 8%.	Pumps based on 400 Ft. Piston Travel per Min. = 700 gals. Obtained at Test

### ENGINE DISPLACEMENT TABLE.

### DOUBLE PUMPS.

	ger Disp ns per F			Pump Rod Correction. Gallons per Revolution.					
Bore of Pump	Strol	ke in Inc	ches.	Diameter of	ches.				
Inches.	7	8	9	Pump Rods.	7	8	9		
3 1/2	1.166	1,888	1,500	1"	0.047	0.054	0.061		
8 5/8	1.951	1.480	1.609	1 1/16	0.058	0.061	0.069		
8 8/4	1.839	1.590	1.721	1 1/8	0.060	0.069	0.078		
8 7/8	1.480	1.684	1.888	1 3/16	0.067	0.077	0.087		
4	1,528	1.740	1.958	1 1/4	0.074	0.085	0,096		
4 1/8	1,620	1.851	2.082	1 5/16	0.081	0.098	0,105		
4 1/4	1,719	1,965	2.211	1 8/8	0.089	0.102	0,118		
4 8/8	1.822	2.088	2.848	1 7/16	0.098	0.112	0.126		
4 1/2	1.928	2,208	2.478	1 1/2	0.107	0.122	0.188		
4 5/8	2,086	2.827	2.618	1 9/16	0.116	0.188	0.150		
4 8/4	2.148	2,455	2.762	1 5/8	0.126	0.143	0.109		
4 7/8	2,968	2,586	2,909	1 11/16	0,136	0.155	0.174		
5	2,880	2,790	8.060	1 8/4	0.146	0.167	0.186		
5 1/8	2,500	2.858	8.215	1					
5 1/4	2.624	2.999	3,874						
5 3/8	2.750	8.143	8,586	Subtract p	•				
5 1/2	2,880	8,291	8,702	rect displace					
5 5/8	8.012	8.442	8.872	For single		_	ise on		
5 8/4	8,147	8.597	4.047	half of resul		-			
5 7/8	8,286	8.755	4.225						
6	8.427	8.917	4.407	1					

Example: Engine with 54-inch pump, 9-inch stroke and 14-inch pump rod.

### From Table above:

Displacement of Plunger = 3.874 gallons. Correction for Rod = 0.188 gallons.

Nominal Displacement = 8.286 gallons

The following table gives the reasonable capacity of several common sizes of fire engines:

### REASONABLE CAPACITIES OF MODERN STEAM FIRE ENGINES.

Bore of Pumps, Inches.	Stroke, Inches.	Capacity, Gallons per Minute
6	9	1,100
5¾	8 or 9	1,000
5¾ 5½	8	900
5¾	8 or 9	850
5	8	750
	8	700
4¾ 4½	7 or 8	600
41/4	7 or 8	550
4	7	` 500

### RATED CAPACITY OF SILSBY ENGINES.

	Nominal Displacement	
Maker's Size.	per Revolution, Gallons.	Rated Capacity, Gallons per Minute
Extra First	1,261	1,000
First	1,141	900
Second	0.952	700
Third	0.804	600
Fourth	0.675	500
Fifth	0.513	400

# TABLE OF HOSE AND NOZZLES FOR TESTING ENGINES, USING SIAMESED LINES.

9

Nors.—Connect Lines to a Deluge Set Provided with a Short Lead of 34. or 4-inch Hose. Use Only Smooth Bore Nozzle and of the Diameter Given. By Regulating One of the Discharge Gates, Pressure can be Kept Nearly Constant and from Three-quarters to Full Capacity Obtained.

Sign	Bore	Reasonable Capacity,	NUMBER AND LENG THE REASONABLE	Number and Length of Lines and Size of Nozele Needed to Deliver the Reasonable Capacity at the Debired Pressure at the Engine.	ILEE OF NOZELE NER	DED TO DELIVER AT THE ENGINE.
	Pump.	Minute.*	100 Pounds.	120 Pounds.	140 Pounds.	160 Pounds.
Double Extra	io	1,100	, ig	, ig	8-150 lines of 8" 8-200 lines of 8" or or 8-150 lines of 895" 8-200 lines of 895"	8-200' lines of 8". or 8-200' lines of 296".
Extra First	*	1,000	S-50 lines of 91%	1-100' line of 294". 1-50' line of 294". 2" Nozzle	2-100 lines of 8" 8" Nozzle	2-150' lines of 24." 2-15' Nozzle
First	346	006	2-50' lines of 234".  156" or 8" Nozzle 1-50' line of 354".  156" or 8" Nozzle 1-50' line of 354".	1–100 line of 81/4" and 1–50 line of 91/4" 17/4" Nozzle	2-100 lines of 21/5 176" Nozzle	8-150 lines of 25%" 17%" Nozzle
Google	۵.	. OBT			2-130' lines of 24'' 14''' Nozzle	2-250' lines of 24''. 14''' Nozzle
n norman	*	<u></u>	2-50' lines of 25."	8-100' lines of 29,8" 156" Nozzle	2-150 lines of 21/6" 19/6" Nozzle	2-350' lines of 295'' 156'' Nozzle
Third	¥9\$	900 00 00 00 00 00 00 00 00 00 00 00 00	1-100' line of 29,5'' and 1-150' line of 29,5'' 19,6'' Nozzle	1-100' line of 25%' and 1-150' line of 25%'' 15'' Nozzle	1-100' line of 29,4" and 1-200' line of 29,4" 13,4" Nozzle	8-860' lines of \$5''' 1}\$'' Nozzle
Fourth	i,	909		2-300' lines of 21/5'' 19/6" Nozzle	8-800 lines of 245" 196" Nozzle	1-100' line of 234" 134" Nozzle ‡
Fifth	:92 86	400	2-100 lines of 214" 114" Nozzle	1–100' line of 814" 114" Nozzle ‡	1-150' line of %%" 114" Nozzle‡	1-900 line of 214" 114" Nozzle ‡

Norm.—If hose has not smoothest lining, shorter lines or a larger nozzle may be required; if hose is slightly larger than given on page 20, it may be necessary to use longer lines or a smaller nozzle. # Single lines; deluge set omitted. \* Based on about 400' piston travel per minute.

Norg.—Connect Line to Nozzie; Bring Engine to Speed and Regulate Discharge Gate; if Desired Pressure Cannot be Obtained, Use Nozzie ½" Smaller or Add Another Length of Hose. TABLE OF NOZZLES FOR TESTING ENGINES, USING SINGLE 50-FOOT LINES OF HOSE.

Sign	Bore	onable scity, na per tote.*	SIZE OF NOZZLE N	Size of Nozzle Needed to Deliver Desired Pressure	THE REASONABLE S AT THE ENGINE.	THE REASONABLE CAPACITY AT THE AT THE ENGINE.
	Pump.	MEAN .	100 Pounds.	190 Pounds.	140 Pounds.	160 Pounds.
First	.7%	8.	<b>.</b> %	Single 50-foot 314" or 8"	Single 50-foot Line of 8" hose	13%" or 13%"
Second		55 55 50 50 50 50 50 50 50 50 50 50 50 5	.g. 13%"	Single 50-foot 15%. 15%.	Single 50-foot Line of 24, bose 15, 15, 15, 15, 15, 15, 15, 15, 15, 15,	15%
Third	23	82 82 82 83	134" or 196" 196"	196" or 114"		19%
Fourth	•	25	1%	19%"	19%"	11%"
Fifth	.3%	9	.7%:	134"	13%"	13/8"

\* Based on about 400' piston travel per minute.

### FIRE STREAM TABLES.

These tables are arranged to show the pressures required at the hydrant or fire engine, while stream is flowing, to maintain nozzle pressures given in the first columns, through various lengths of 2½-, 3- and 3½-inch rubber lined hose in single lines and two lines of 2½-inch hose siamesed.

The pressure at the hydrant or fire engine is that indicated by a gage attached to the hydrant or fire engine while the stream is flowing. The pressure at the nozzle is that indicated by a Pitot gage held in the stream.

The hydrant (or engine) pressures are obtained by adding to the nozzle pressure the friction loss in the hose, and also the small additional loss in the hydrant outlet or engine discharge.

Friction losses in hose are based on tests of best quality rubber-lined fire hose and are for 100-foot lengths measured without pressure applied. Diameters of hose, as measured under 75 pounds pressure, assumed as the average working condition, were as follows: For nominal 2½-inch, 2.575 or about 2 9/16 inches; for nominal 3-inch, 3.125 or 3½ inches; for nominal 3½-inch, 3.685 or about 3 11/16 inches.

The smoothness of the lining has a very considerable effect on the friction loss, some samples tested showing losses 50 per cent. in excess of those given. A slight variation in diameter also produces a marked difference in friction loss; in the case of 2½-inch hose, a variation of 1/16 inch in diameter will result in 10 per cent. difference in loss. If properly beveled 2½-inch couplings are used on 3-inch hose, the loss of pressure due to them will be less than 5 per cent. of that gained by the use of the larger hose. For instance, for a flow of 300 gallons per minute, the loss in 2½-inch hose will be about 21 pounds, in 3-inch hose with 3-inch couplings about 8 pounds, and in 3-inch with 2½-inch couplings about 8½ pounds.

For siamesed lines, an allowance was made for the loss in the siamese connection and for 20 feet of 3½-inch lead hose.

The pressures given are for the nozzle at the same elevation as the hydrant or engine discharge outlet. Add or subtract I pound to the pressure given for each 2 I/3 feet difference in elevation. The arrangement of the table allows a comparison to be readily made of the results obtainable with 3-inch hose and siamesed lines against single lines of 2½-inch hose.

### EFFECTIVE REACH OF FIRE STREAMS.

SHOWING THE DISTANCE IN FEET FROM THE NOZZLE AT WHICH STREAMS WILL DO EFFECTIVE WORK WITH A MODERATE WIND BLOWING. WITH A STRONG WIND THE REACH IS GREATLY REDUCED.

=	1 .				ZE OF N	Iorr P				
zzle.	1-I1	nch.	1-I	nch.	1½-I		1 <b>∦</b> −I	nch.	ı≟-I	nch.
Pressure at Nozzle.	Vertical Dis- tance, Feet.	Horizontal Dis- tance, Feet.	Vertical Dis- tance, Feet.	Horizontal Distance feet.	Vertical Dis- tance, Feet.	Horizontal Dis- tance, Feet.	Vertical Dis- tance, Feet.	Horizontal Dis- tance, Feet.	Vertical Dis- tance, Feet.	Horizontal Dis- tance, Feet.
20	35	37	36	38	36	39	36	40	37	42
25	43	42	44	44	47	46	45	47	46	49
80	51	47	52	50	52	52	53	54	54	56
85	58	51	59	54	59	58	59	59	62	62
40	64	55	65	59	65	62	66	64	69	66
45	69	58	70	63	70	65	72	68	74	71
50	73	61	75	66	75	69	77	72	79	75
55	76	64	79	69	80	72	18	75	82	78
60	79	67	83	72	84	75	85	77	87	80
65	82	70	86	75	87	78	88	79	90	81
70	85	72	88	77	90	8o	91	82	92	84
75	87	74	90	79	92	82	93	84	94	86
80	89	76	92	81	94	84	95	86	96	88
85	91	78	94	83	96	87	97	88	99	90
90	92	80	96	85	98	89	99	90	100	91

NOTE.—Nozzle pressures are as indicated by Pitot tube. The horizontal and vertical distances are based on experiments by Mr. John R. Freeman, Transactions, Am. Soc. C. E., Vol. XXI.

### FRICTION LOSS IN FIRE HOSE.

# Based on Tests of Best Quality Rubber Lined Fire Hose.\*

Flow, Gallons per Minute.	10	o FEE	T OF	IN EACH Hose, Q. Inch.	, Gallons per Minute.	PRESSURE LOSS IN EACH 100 FEET OF HOSE, POUNDS PER SQ. INCH.			
Flow, G.	2½° Hose.	3" Hose.	3½″ Hose.	2 Lines of 2½° Siamesed.	Flow, Gallons Minute.	3° Hose.	3½° Hose.	2 Lines of $2\frac{1}{2}$ . Siamesed.	
140	5.2	2.0	0.9	1.4	525	23.2	10.5	16.6	
160	6.6	2.6	1.2	1.9	550	25.2	11.4	18.1	
180	8.3	3.2	1.5	2.3	575	27.5	12.4	19.0	
200	10.1	3.9	1.8	2.8	600	29.9	13.4	21.2	
220	12.0	4.2	2.1	3.3	625	32.0	14.4	23.0	
240	14.1	5.4	2.5	3.9	650	34 · 5	15.5	24.8	
260	16.4	6.3	2.9	4 · 5	675	37.0	16.6	26.5	
280	18.7	7.2	3.3	5.2	700	39.5	17.7	28.3	
300	21.2	8.2	3.7	5.9	725	42.3	18.9	30.2	
320	23.8	9.3	4.2	6.6	750	45.0	20. I	32.2	
340	26.9	10.5	4.7	7.4	775	47.8	21.4	34.2	
360	30.0	11.5	5.2	8.3	800	50.5	22.7	36.2	
380	33.0	12.8	5.8	9.2	825	53.5	24.0	38.4	
400	36.2	14.1	6.3	10.1	850	56.5	25.4	40.7	
425	40.8	15.7	7.0	11.3	875	59.7	26.8	43. I	
450	45.2	17.5	7.9	12.5	900	63.0	28.2	45.2	
475	50.0	19.3	8.7	.13.8	1,000	76.5	34.3	55.0	
500	55.0	21.2	9,5	15.2	1,100	91.5	41.0	65.5	

<sup>\*</sup>Rough rubber lining is liable to increase the losses given in the table as much as 50 per cent.

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## DISCHARGE TABLE FOR SMOOTH NOZZLES.

#### NOZZLE PRESSURE MEASURED BY PITOT GAGE.

Nozzle	No	EZLE D	IAM. IN		8,	ll I	Noz	zle Di	AM. TN	Inche	a
Pressure	1	11/6	11/4	136	11/6	Nozzle Pressure	1	11/6	11/4	136	134
n lbs. per – sq. inch.	Ga	llons	per mi	nute.		in lbs. per sq. inch.	G	allons	per M	inute.	
5	66	84	108	125	149	60	229	290	85/7	484	517
6	72	92	118	187	168	62	288	295	868	441	595
7	78	99	122	148	176	64	287	299	869	448	588
8	84	106	181	158	188	66	240	804	875	455	549
•	89	112	189	168	200	68	944	808	881	462	550
10	98	118	146	177	211	70	247	818	886	469	558
12	102	180	160	194	281	72	251	818	891	475	566
14	110	140	178	210	249	74	254	822	897	488	574
16	118	150	185	224	267	76	258	826	402	488	268
18	125	159	196	287	288	78	261	. 880	407	494	589
20	182	167	206	250	298	80	264	885	418	500	596
22	189	175	216	268	818	82	<b>268</b>	889	418	507	604
24	145	188	296	275	827	84	271	848	428	518	611
26	151	191	285	286	840	86	274	847	428	519	618
28	157	198	244	297	858	88	277	851	488	525	626
80	162	905	258	807	868	90	280	855	488	581	688
82	167	212	261	817	877	92	283	859	448	587	640
84	179	218	269	827	389	94	996	868	447	548	647
86	177	224	277	886	400	96	289	867	452	549	654
\$8	182	281	285	845	411	98	292	870	456	554	660
40	187	287	292	854	422	100	295	874	461	560	667
42	198	948	299	868	489	105	808	888	478	574	688
44	196	248	806	872	448	110	810	892	484	588	699
46	900	254	818	<b>88</b> 0	452	115	817	401	495	600	715
48	205	259	820	888	462	120	894	410	505	618	<b>78</b> 0
50	209	265	826	896	472	125	881	418	516	696	745
52	218	270	838	404	481	180	887	497	596	<b>688</b> -	760
54	217	275	889	419	490	185	848	485	586	650	776
56	221	280	845	419	499	140	<b>85</b> 0	448	546	662	789
58	225	285	851	496	508	145	856	450	556	674	808
60	229	290	857	484	517	150	862	458	565	686	817

#### DISCHARGE TABLE FOR SMOOTH NOZZLES.

#### NOZZLE PRESSURE MEASURED BY PITOT GAGE.

Nozzle Pressure in 1bs. per	No.	EELS D	14m. in 1%	8 Inchi	9 <u>14</u>	Nozzle Pressure	Noz 156	ZLB D1 134	1% 1%	INCHE 2	8. 21/4
sq. inch.	G	allons	per M	inute.		in lbs. per sq. inch.	G	allons	per Mi	nute.	
5	175	208	284	266	887	60	607	704	810	990	116
6	192	2028	256	292	869	62	617	716	828	986	118
7	207	241	277	815	899	64	627	727	886	951	120
8	222	257	296	886	497	66	686	788	850	965	122
•	285	278	814	857	452	68	646	750	862	980	194
10	248	288	880-	876	477	70	655	761	875	994	126
12	271	315	862	412	522	72	665	771	887	1008	127
14	298	840	891	,445	564	74	674	788 .	900	1028	129
16	818	364	418	475	608	76	688	798	911	1086	181
18	888	886	444	504	640	78	692	808	924	1050	188
20	850	407	468	582	674	80	700	818	985	1068	184
22	867	427	490	557	707	82	709	828	946	1076	186
24	884	446	512	582	789	84	718	888	959	1089	188
26	400	464	588	606	769	86	796	848	970	1102	189
28	415	481	554	629	799	88	785	858	981	1115	141
80	429	498	572	661	896	90	748	862	992	1128	148
82	448	514	591	<i>,</i> 078	854	92	751	879	1002	1140	144
84	457	580	610	698	880	94	759	881	1012	1159	146
86	<b>47</b> 0	546	627	718	905	96	767	890	1022	1164	147
88	488	561	645	788	980	98	775	800	1082	1176	149
40	496	575	661	752	954	100	788	909	1048	1189	150
42	508	589	678	770	978	105	808	982	1070	1218	154
44	590	608	604	788	1000	110	822	954	1095	1247	158
46	581	617	710	806	1021	115	840	975	1190	1975	161
48	548	680	725	894	1048	120	<b>85</b> 8	996	1144	1808	168
50	554	648	740	841	1065	125	876	1016	1168	1899	166
52 ·	565	656	754	857	1087	180	898	1086	1191	1856	171
54	576	668	769	878	1108	185	910	1056	1218	1882	170
56	586	680	788	889	1199	140	997	1076	1285	1407	178
\$8	596	692	796	905	1149	145	944	1095	1957	1432	181
60	607	704	810	990	1168	150	960	1114	1279	1456	184

ssure I by . ge.	Discharge, Gallons per Minute.	PRESSURES REQUIRED AT HYDRANT O MAINTAIN NOZZLE PRESSURES GIVE LENGTHS OF BEST QUALIT								
Nozzle Pressure Indicated by Pitot Gage.	charge, (				Sin	gle 21	-inch	Lines.	:	
No. Ir	Discl	100 Feet.	200 Feet.	300 Feet.	400 Feet.	<b>500</b> Feet.	600 Feet.	700 Feet.	800 Feet,	
20	132	25	30	35	39	44	49	53	58	
25	148	31	37	43	49	55	60	66	72	
80	162	38	44	51	58	65	72	78	85	
85	175	44	52	59	67	75	83	91	98	
40	187	50	59	68	77	86	94	103	112	
45	198	56	66	76	86	96	106	115	125	
50	209	62	73	84	95	106	117	128	139	
55	219	68	80	92	104	116	128	140	152	
60	229	75	88	101	114	127	140	153	166	
65	238	81	95	109	123	137	151	165	179	
70	247	87	102	117	132	147	162	177	192	
75	256	93	109	125	141	157	173	189	205	
80	264	99	116	133	150	167	183	200	217	
85	272	105	123	141	159	177	195	212	230	
90	280	111	130	149	167	186	205	224	243	
95	287	117	137	157	177	196	216	236	256	
100	295	123	144	165	185	206	227	247	268	

in F	Engii Irst nd 3-1	Colt	JMN,	THRO	UGH	FLOWI VARIO IOSE.		o	ssure   by ge.
		Sing	gle 3-iı	nch Li	nes.		o 2½-i: s Siam		Nozzle Pressure Indicated by Pitot Gage.
1,000 Feet.		800 Feet.	1,000 Feet.	1,200 Feet.	1,500 F <b>e</b> et.	1,000 Feet.	1,500 Feet.	2,000 Feet.	No.
68	77	35	39	42	48	33	40	46	20
84	95	43	48	52	59	41	49	57	25
99	112	52	57	62	70	49	59	68	80
114	130	60	66	72	81	57	68	79	85
130	148	68	75	82	92	65	78	90	40
145	165	77	84	92	103	72	86	99	45
160	182	85	93	102	114	80	95	110	50
175	199	93	102	112	125	88	105	121	55
192	218	102	112	122	137	96	114	132	60
207	235	110	121	131	148	103	122	141	65
222	252	118	130	141	159	111	132	152	70
237	269	127	139	151	170	120	142	164	75
251	285	135	148	161	181	128	151	175	80
266	302	143	156	170	191	135	159	184	85
280		151	165	180	202	143	169	195	90
295		158	173	189	211	150	177	204	95
310		167	183	199	223	157	186	215	100

Nozzle Pressure Indicated by Pitot Gage.	Discharge, Gallons per Minute.	PRESSURES REQUIRE NOZZLE PRES							EN :		IRST
icated b Gage	charge, Gal per Minute.				Single	2 <b>}-</b> i	nch I	Lines.			
Ind	Dis	Feet.	Feet.	300 Feet.	400 Feet.	Feet.	600 Feet.	700 Feet.	800 Feet.	1,000 Feet.	1,200 Feet.
20	167	28	35	42	49	56	64	71	. 78	92	107
25	187	35	44	53	62	71	79	88	97	115	133
80	205	42	52	63	73	84	95	105	116	137	158
85	221	49	61	73	85	97	110	122	134	158	183
40	237	55	69	83	96	110	124	138	151	179	206
45	251	62	77	93	108	123	139	154	169	200	230
50	265	69	86	103	120	137	154	171	188	222	256
55	277	76	94	112	131	149	168	186	204	241	278
60	290	83	103	123	143	163	183	203	223	263	304
65	301	89	111	132	154	175	197	218	240	283	326
70	313	96	119	142	165	188	211	234	257	303	
75	324	103	128	152	177	202	227	252	276	325	
80	335	110	136	162	188	215	241	267	294	ļ	
85	345	116	144	171	199	226	254	282	309	····	
90	355	123	152	181	210	240	269	298	327	<b>,</b>	
95	365	130	160	191	222	252	283	314	•		
100	374	136	168	201	233	269	297	329	9		

Engine, while stream is flowing, to maintain Column, through various Lengths of Best 3-inch Rubber Lined Hose.

essur Pitot	es	Line	-inch	wo 2	T,	- 1		<del></del>	. ,			
e Pre			mese				es. 	Line	3-inct	igle 3	Sir	
Nozzle Pressur cated by Pitot	1,800 Feet.	1,500 Feet.	1,200 Feet.	1,000 Feet.	800 Feet	1,800 Feet.	1,500 Feet.	1,200 Feet.	1,000 Feet.	800 Feet.	600 Feet.	Feet.
20	60	53	46	42	38	71	62	54	48	43	37	32
25	70	63	55	50	45	87	77	67	60	53	46	40
80	82	74	65	59	53	103	91	79	71	63	55	47
85	96	86	76	69	62	121	107	93	83	74	65	55
40	108	97	<b>8</b> 6	78	70	137	121	105	95	84	73	63
45	121	108	95	87	79	153	135	118	106	94	82	70
50	135	121	107	98	88	169	150	130	117	104	91	78
55	147	132	117	107	96	185	164	142	128	114	100	86
60	160	143	127	116	105	201	178	155	139	124	109	93
65	174	156	138	126	114	217	192	167	151	134	117	101
70	186	167	148	135	122	233	206	180	162	144	126	108
75	198	178	157	144	130	249	221	192	173	1 54	135	116
80	210	189	167	153	138	267	236	206	185	165	144	124
85	224	201	178	163	147	281	249	217	195	174	153	131
90	237	212	188	172	1 56	297	263	229	207	184	161	1 39
95	249	224	198	181	164	313	277	242	218	194	170	146
100	261	235	208	190	172		290	253	228	203	178	154
11	1			•	13	1	1	•	1	τ .	I	

## 1 1/4-INCH SMOOTH NOZZLE.-

Nozzle Pressure Indicated by Pitot	Gallons iute.	PR					AT EN I	N F	RANT IRST }- AN	Col	
ozzle Pressure Ir dicated by Pitot Gage.	Discharge, Gall per Minute.			ļ	Singl	e 2 <b>1-</b> i	inch l	Lines	<b>.</b>		
Nozzl dic	Disc	Feet.	Feet.	300 Feet.	400 Feet.	500 Feet.	Feet.	700 Feet.	Feet.	1,000 Feet.	1,200 Feet.
20	206	32	42	53	64	75	85	96	107	128	149
25	230	40	53	66	79	92	105	118	131	1 58	184
80	253	48	63	79	95	110	126	142	157	189	220
85	273	55	73	91	109	127	145	163	181	217	253
40	292	63	83	104	124	144	165	185	206	246	287
45	309	70	93	116	138	161	183	206	229	274	319
50	326	78	103	128	153	178	203	228	253	303	
55	342	86	113	140	167	194	222	249	276	330	
60	357	93	123	1 52	182	211	241	270	300		
65	372	101	133	164	196	228	260	292	323		
70	386	108	142	176	210	244	278	312			
75	399	116	1 52	188	224	261	297	333			
80	413	124	163	201	240	279	318				
85	425	131	172	213	254	295					• • • •
90	438	139	182	225	269	312					•••
95	449	146	191	236	282	327	•••		•••		
100	461	153	201	248	295	••••					
		1					1	!		!	

Engine, while stream is flowing, to maintain Nozzle through various Lengths of Best Quality Rubber Lined Hose.

	Si	ngle :	3-incl	Lin	es.				2½-i Siam		Lines		Nozzle Pressu dicated by Pitot
400 Feet.	600 Feet	800 Feet.	1,000 Feet.	1,200 Feet.	1,500 Feet.	1,800 Feet.	600 Feet.	Scet.	1,000 Feet.	1,200 Feet.	1,500 Feet.	1,800 Feet.	Nozzl dicate
37	46	54	62	70	83	95	39	45	51	57	67	76	20
47	57	67	77	87	102	117	48	55	62	70	<b>8</b> 0	91	25
56	68	81	93	105	123	142	57	66	74	83	96	109	80
65	79	92	106	I 20	141	161	66	76	86	95	110	125	85
74	89	105	120	136	159	183	75	87	99	110	127	144	40
83	100	117	135	152	178	204	84	96	109	121	140	158	45
91	111	130	149	168	197	226	93	107	121	135	155	176	50
100	121	142	163	184	216	247	102	117	132	147	169	192	55
109	132	1 5.5	178	. 201	235	270	111	128	144	160	185	210	60
118	143	167	192	217	254	291	120	137	155	173	199	225	65
127	154	180	. 206	233	272		129	147	166	185	213	241	70
136	164	192	. 220	248	290		137	157	177	197	227	257	75
145	175	205	235	265	<b> </b>		147	169	190	212	244	276	80
153	184	21.6	247	279			156	179	201	224	258	292	85
162	195	228	. 261	295			.165	189	213	237	273	309	90
170	205	240	27.5	<b> </b> .		• . •••	.173	198	223	248	286	323	95
179	215	252	. 288				.182	208	235	261	300		100

#### 1 3/8-INCH SMOOTH NOZZLE.-

Nozzle Pressure Indi- cated by Pitot Gage.	Discharge, Gallons per Minute.	PRESSURES REQUIRED AT HYDRANT OR FIRE NOZZLE PRESSURES GIVEN IN FIRST QUALITY 21- AND										
e Pressi I by Pito	charge, per Min		5	Single	2 <del>1</del> - i	nch	Lines	<b>3.</b>				
Nozzl   cated	Disc	Fe 2	200 Feet.	300 Feet.	400 Feet.	Feet,	Feet.	Feet.	Feet 8	200 Feet.	400 Feet.	
20	250	37	52	68	83	98	113	128	144	34	45	
25	<b>28</b> 0	46	64	83	102	121	139	158	177	41	56	
80	307	55	77	99	121	144	166	188	210	50	67	
85	331	64	89	115	140	166	191	217	242	58	78	
40	354	73	102	131	160	189	218	247	276	67	89	
45	376	81	114	146	178	211	243	275	307	74	99	
50	396	90	125	161	196	222	257	293	328	82	109	
55	415	99	137	176	215	254	292	331		90	121	
60	434	107	149	191	233	276	318			98	131	
65	451	116	161	206	251	297				106	141	
70	469	125	173	222	270	319				114	152	
75	485	134	185	237	289	• • • • •				122	162	
- 80	500	142	196	251	305					130	172	
85	516	151	209	267	325					138	183	
90	531	159	220	281						146	194	
. 95	546	168	232	297						153	203	
100	560	177	244	312	• • •.•	••••		••••		162	215	

Engine, while stream is flowing, to maintain Column, through various Lengths of Best 3-inch Rubber Lined Hose.

Pressu by Pitot	ed.	ımes	es Sia	Line	-inch	vo 2	Tv	es.	Line	-incl	ngle 3	Siı
Nozzle P	1,800 Feet.	1,500 Feet	1,200 Feet.	1,000 Feet.	800 Feet.	600 Feet.	400 Feet.	1,500 Feet.	1,200 Feet.	1,000 Feet.	800 Feet.	600 Feet.
20	96	84	71	63	54	46	37	109	92	80	68	57
25	119	104	88	78	67	57	46	135	113	99	85	70
80	143	124	106	93	81	68	56	161	135	118	101	84
85	165	143	122	108	94	80	65	187	157	137	117	97
40	186	162	138	122	106	90	74	214	180	157	134	112
45	209	182	1 55	137	119	101	83	238	200	175	150	125
50	230	201	171	151	131	111	92	267	220	192	164	137
55	252	219	187	165	144	122	100	288	242	212	182	151
60	273	238	203	180	156	133	109		262	229	196	163
65	294	257	219	194	168	143	118		282	247	212	177
70	317	277	236	209	182	155	128		303	265	227	189
75		295	252	223	194	165	137		<b> </b>	283	243	203
80		312	266	236	206	175	145		•••	300	257	215
85		331	282	250	218	1.86	1.53	<b> </b>			274	229
90			298	264	230	196	162		. <b></b> .	ļ 	289	241
95			313	277	241	206	170	<b> </b>		ļ	304	254
100			329	291	254	217	179					26.7

# 1 1/2-INCH SMOOTH NOZZLE.—

FIRE IRST AND	N F	EN I	YDR GIV QUAI	URES	ED A	UIRE LE P	REQ		ESSU	PR	Gallons ute.	Nozzle Pressure Indi- cated by Pitot Gage.
ingle	S				Lines	nch l	e 2½-i	Single	- 3		Discharge, Gall per Minute.	Pressu
600 Feet.	400 Feet.	200 Feet.	800 Feet.	700 Feet.	600 Feet.	500 Feet.	400 Feet.	300 Feet.	200 Feet.	reet.	Nozzle Nozzle Se Cated Property Property Property Nozzle Prope	Nozzle cated b
71	55	39	191	170	149	128	107	86	65	44	298	20
88	68	48	236	210	184	158	132	106	80	54	333	25
105	81	58	280	250	219	188	157	126	95	65	365	30
122	94	67	322	287	251	216	181	145	110	75	394	35
139	107	76		327	286	246	206	166	126	85	422	40
155	120	85			320	275	230	185	141	96	447	45
171	133	95			74.44	304	254	205	155	106	472	50
187	145	104				332	278	224	170	116	494	55
203	158	113					301	242	184	126	517	60
218	170	122					324	261	198	136	537	65
235	183	131		1000				281	213	146	558	70
251	196	140						299	228	156	578	75
267	208	149						318	242	166	596	80
282	220	158				****		337	257	176	614	85
298	233	167							272	187	633	90
314	245	176			,,,,				286	197	650	95
	257	185							300	207	667	100

160 212 264

Nozzle Pressure Indi- cated by Pitot Gage.	Discharge, Gallons per Minute.	PR	ESSU				SSUR	HYDI ES GI QUAI	VEN	OR I	
Pressu by Pito	arge, C r Minu	-	Single	2 ½-i	nch l	Lines	.		Sin	gle 3	-inch
Nozzle cated b	Dischai per	Feet.	200 Feet.	300 Feet.	400 Feet.	500 Feet.	600 Feet.	200 Feet.	400 Feet.	600 Feet.	800 Feet.
20	350	52	80	108	136	165	193	46	68	90	112
25	392	65	100	135	170	205	240	57	84	111	138
80	429	77	118	160	201	242	284	68	100	132	164
85	463	89	136	184	231	279	326	78	115	152	189
40	496	101	155	208	262	316		89	131	173	215
45	525	113	173	233	293			100	146	193	239
50	554	125	192	258	324	<b> </b> .		111	162	214	265
55	581	137	210	282	<b> </b>	<b> </b>		121	178	234	290
60	607	149	228	306	ļ			132	193	254	
65	631	162	246	330	<b></b>			143	209	275	
70	655	173	263		<b> </b> .			153	223	294	
75	678	184	281				<b> </b>	163	237	312	
80	700	197	299				<b> </b>	174	253		<b> </b>
85	722	209	317		<b> </b> .	<b> </b>		184	269		
90	743	220	· · · ·			<b> </b>		195	284	ļ	,
95	763	232	ļ	<b></b> .		<b> </b> .		205	299		
100	783	244	ļ			<b></b>		216	314	····	

Engine, while stream is flowing, to maintain Column, through various Lengths of Best 3-inch Rubber Lined Hose.

Lines.	Lines. Two 2½-inch Lines Siamesed.										
1,000 Feet.	1,200 Feet.	200 Feet.	400 Feet.	600 Feet.	800 Feet.	1,000 Feet.	1,200 Feet.	1,500 Feet.	1,800 Feet.	Nozzle Pre Indicate by Pitot G	
134	156	37	53	68	84	100	115	139	162	20	
165	192	47	66	85	104	123	143	171	200	25	
196	228	56	79	102	125	148	171	205	240	80	
226	263	65	91	117	144	170	197	236	276	85	
257	299	74	104	134	164	194	224	269	314	40	
286	•••	82	116	149	182	215	248	298		45	
		91	128	165	202	239	275	331		50	
		100	140	181	221	261	301			55	
		109	153	196	240	283	327			60	
		118	164	211	258	305				65	
.••••		126	176	226	276	<b>32</b> 6				70	
		135	189	242	295					75	
• • • •	<b> </b>	144	201	258	314					80	
		153	213	273						85	
. • • • •		162	225	289						90	
		170	237	303						95	
••••		179	249	319			<u> </u>			100	

Nozzle Pressure Indi- cated by Pitot Gage.	Discharge Gallons per Minute.	Pressures Required at Hydrant or Fire Nozzle Pressures given in First Quality 2½- and												
Pressi by Pite	charge Gall per Minute.	Sing	le 21	-in. L	ines.				Sin	gle 3	-inch			
Nozzle cated	Disc	100 Feet.	200 Feet.	300 Feet.	400 Feet.	100 Feet.	200 Feet.	300 Feet.	400 Feet.	500 Feet.	600 Feet.			
20	407	63	100	138	175	40	55	71	86	101	116			
25	455	77	123	169	215	49	67	84	102	120	138			
80	498	91	145	199	253	58	79	100	121	142	163			
85	538	106	169	231	294	68	92	117	141	166	190			
40	575	120	191	262	333	77	104	132	159	187	215			
45	609	135	215	294		87	118	149	180	211	241			
50	643	150	237	325		96	130	164	199	233	267			
<b>5</b> Š	674	164	<b>25</b> 9			105	142	179	216	254	291			
60	704	177	280			114	I 54	194	234	274	314			
65	732	191	302			123	166	209	252	296	<b>.</b>			
70	76 I	206	325			133	180	227	273		···•			
75	787	220				143	192	242	291					
80	813	234				152	204	257	309					
85	838	247				160	215	270						
90	862	261				169	228	286						
95	885	274				178	240	301		· · · · ·				
100	909					188	253	317			, .			

Engine, while stream is flowing, to maintain Column, through various Lengths of Best 3-inch Rubber Lined Hose.											Nozzle Pressure Indi- cated by Pitot Gage.
Line	s.		T	wo 2	}-inch	Lin	e <b>s</b> Sia	mese	d.		e Pressur by Pitot
800 Feet.	1,000 Feet.	Feet.	200 Feet.	300 Feet.	400 Feet,	500 Feet.	600 Feet.	800 Feet.	1,000 Feet.	1,200 Feet.	Nozzle
147	177	33	43	53	64	74	84	105	125	146	20
173	209	40	53	65	78	91	103	128	1 54	179	25
205	247	49	64	79	94	110	125	155	185	215	80
239	288	56	74	91	109	126	143	178	213	248	85
270	325	64	84	103	123	143	162	201	241	280	40
<b>3</b> 03		73	95	117	139	161	183	227	271	315	45
		80	104	128	152	177	201	249	297	· • • ·	50
		88	114	140	167	193	219	272	324		55
• • • •		96	125	153	182	210	239	296			60
		104	134	165	195	226	257	318			65
		111	144	177	210	243	275				70
• • •		118	153	188	223	258	293			••••	75
		127	164	201	239	276	313				80
••••		1 35	174	214	253	293					85
• • • •		142	183	225	266	308			• • • •		90
		150	194	237	281						95
• • • •		158	204	. 250	296			••••			100

Nozzle Pressure Indi- cated by Pitot Gage.	Discharge, Gallons per Minute.	Pressures Required at Hydrant or Fire Nozzle Pressures given in First Quality 21- and										
le Press I by Pit	charge, per Min	Si	ingle 2 ch Lin	es.		S	ingle	3-inch	Lines.			
Nozz	Disc	100 Feet.	200 Feet.	300 Feet.	100 Feet,	200 Feet.	300 Feet.	400 Feet.	500 Feet.			
20	532	90	152	214	52	76	100	124	148			
25	594	111	187	263	65	94	123	152	182			
80	651	132	222	312	77	112	147	181	216			
85	703	152	255		89	129	169	209	249			
40	752	173	290		102	147	193	238	283			
45	797	193	323		113	163	213	263	314			
50	841	214			126	182	237	293	•			
55	881				138	199	260	321	· • • •			
60	920				150	216	282	<b> </b>				
65	958				162	233	304	ļ				
70	994				175	251	327	<b> </b>				
75	1,029				187	268	<b> </b> .	<b> </b>				
80	1,063				199	285	 					
85	1,095				211	302		<b></b>	<b></b>			
90	1,128				223	319	<b></b> .	<b> </b>				
95	1,158				235	335	<b></b> .					
100	1,189				247							

ENGINE, WHILE STREAM IS FLOWING, TO MAINTAIN COLUMN, THROUGH VARIOUS LENGTHS OF BEST 3-INCH RUBBER LINED HOSE.

#### 1 1/4-INCH SMOOTH NOZZLE.-3 1/2-INCH HOSE.

Nozzle Pressure Indi- cated by Pitot Gage.	Discharge, Gallons per Minute.	DRA STR TAI: FIR: LEN	EAM N No ST Co	OR I IS OZZLE OLUM	FIRE FLOV PRI IN, T BE	VING ESSUI HRO	GINE, TO RES O UGH QUAI	, WE MAGIVEN VARI LITY		Nozzle Pressure Indi- cated by Pitot Gage.
Nozzle	Disch	600 Feet.	700 Feet.	800 Feet.	900 Feet.	1,000 Feet.	1,200 Feet.	1.500 Feet.	1,800 Feet.	Nozzle cated
20	206	32	34	36	37	39	43	49	55	20
25	230	39	42	44	46	48	53	60	67	25
80	253	47	49	52	55	58	63	71	79	80
85	273	54	57	60	64	67	73	82	91	85
40	292	62	65	69	72	76	83	93	104	40
45	309	69	73	77	81	85	93	104	116	45
50	326	77	18	85	90	94	102	115	128	50
55	342	84	89	94	99	103	112	126	141	55
60	357	92	97	102	107	112	122	137	153	60
65	372	99	105	110	116	121	1 32	149	165	65
70	386	107	113	118	124	130	142	160	177	70
75	399	114	I 20	127	133	139	152	171	190	75
80	. 413	122	128	135	142	148	162	182	202	80
85	425	128	135	142	149	156	170	191	212	85
90	438	136	143	151	158	165	180	202	225	90
95	449	143	151	159	167	175	190	214	237	95
100	461	151	159	167	175	184	200	225	249	100

# 1 3/8-INCH SMOOTH NOZZLE.—3 1/2-INCH HOSE.

Nozzle Pressure Indi- cated by Pitot Gage.	Discharge, Gallons per Minute.	PRESSURES REQUIRED AT HYDRANT OR FIRE ENGINE, WHILE STREAM IS FLOWING, TO MAINTAIN NOZZLE PRESSURES GIVEN IN FIRST COLUMN, THROUGH VARIOUS LENGTHS OF BEST QUALITY 3½-INCH RUBBER LINED HOSE.										
Nozz	Dis	400 Feet.	Feet. Feet. 1,500 Feet. 1,500 Feet. 1,500 Feet.									
20	250	31	34	36	39	41	47	52	60	67	20	
25	280	39	42	45	49	52	59	65	75	85	25	
80	307	46	50	54	58	62	70	78	89	101	80	
85	331	54	58	63	67	72	81	90	103	117	85	
40	354	61	66	71	76	81	91	101	116	131	40	
45	376	69	74	80	85	91	102	113	130	147	45	
50	396	76	82	88	95	101	113	126	144	163	50	
55	415	84	90	97	104	111	124	138	158	179	55	
60	434	91	98	106	113	121	135	150	172	195	60	
65	451	98	106	114	122	130	146	161	185	209	65	
70	469	106	114	123	131	140	1 57	174	199	225	70	
75	485	113	122	131	140	149	167	185	212	239	75	
80	500	1 20	130	140	149	159	178	197	226	255	80	
85	516	127	138	148	158	168	188	208	239	269	85	
90	531	135	146	156	167	178	199	221	253	285	90	
95	546	142	153	165	176	187	209	232	266	299	95	
100	560	150	161	173	185	197	2 <b>2</b> 0	244	279	315	100	
	. '			•			-	,		,		

## 1.1/2-INCH SMOOTH NOZZLE.-3 1/2-INCH HOSE.

Nozzle Pressure Indi- cated by Pitot Gage.	Discharge, Gallons per Minute.	DRA STR TAI FIR LEI	EAM IN NO ST C NGTH IN RI	IS OZZLI OLUM S OI	FIRE FLOV E PRI IN, TI E BE	Enc WING ESSUI HROU EST (	GINE,	WH MA IVEN VARIO	IN- IN	Nozzle Pressure Indi- cated by Pitot Gage.	
Z 0	I	F. P. 2	400 Feet	Fe		P.e	1,2 Fe	1,500 Feet.	1,8 Fe	Z	
20	298	28	36	43	50	58	65	76	87	20	
25	333	35	44	53	62	71	80	93	107	25	
<b>80</b> .	365	42	53	63	74	85	96	112	128	80	
85	394	49	61	73	86	98	111	129	148	85	
40	422	55	69	83	97	111	125	146	167	40	
45	447	62									
50	472	69	86	103	121	138	155	181	207	50	
55	494	76	94	113	132	151	170	198	226	55	
60	517	82	102	123	143	163	183	214	244	60	
65	537	89	111	133	154	176	198	231	263	65	
70	558	96	119	143	166	189	213	248	283	70	
75	578	103	128	153	178	203	228	265	303	75	
80	596	109	136	162	188	215	241	281		80	
85	614	116	144	172	200	228	256	298		85	
90	633	123	152	182	211	241	271	<b> </b>		90	
95	650	129	160	191	222	253	284	<b> </b>		95	
100	667	136	168	201	233	265	298	<b> </b>		100	

## 1 3/4-INCH SMOOTH NOZZLE.—3 1/2-INCH HOSE.

Nozzle Pressure Indi- cated by Pitot Gage.	Discharge, Gallons per Minute.	FIR ING GIV VAI 31-1	PRESSURES REQUIRED AT HYDRANT OR FIRE ENGINE, WHILE STREAM IS FLOWING, TO MAINTAIN NOZZLE PRESSURES GIVEN IN FIRST COLUMN, THROUGH VARIOUS LENGTHS OF BEST QUALITY 31-INCH RUBBER LINED HOSE.											
											Nozzle Pressure Indi-			
20	407	28	35	41	48	54	61	74	87	101	20			
25	455	35	43	51	59	67	75	91	107	123	25			
80	498	41	51	60	70	79	89	108	127	146	80			
85	538	48	<b>5</b> 9	70	81	92	103	124	146	168	85			
40	575	55	67	80	92	105	117	142	167	191	40			
45	609	62	75	89	103	117	131	158	186	213	45			
50	643	68	84	99	115	130	145	176	206	237	50			
55	674	75	92	109	125	142	159	192	225	259	55			
60	704	82	100	118	136	154	172	208	244	280	60			
65	732	89	108	127	147	166	186	224	263	302	65			
70	761	95	116	137	158	178	199	241	282		70			
75	787	102	124	146	168	190	212	257	301		75			
80	813	109	132	156	179	203	226	273	320		80			
85	838	115	140	165	190	214	239	289	<b> </b>		85			
90	862	122	148	174	200	227	253	305	<b> </b>		90			
95	885	128	156	183	211	238	266				95			
100	909	135	164	193	222	251	280		<b> </b>		100			

## 1 5/8-INCH SMOOTH NOZZLE.—3 1/2-INCH HOSE.

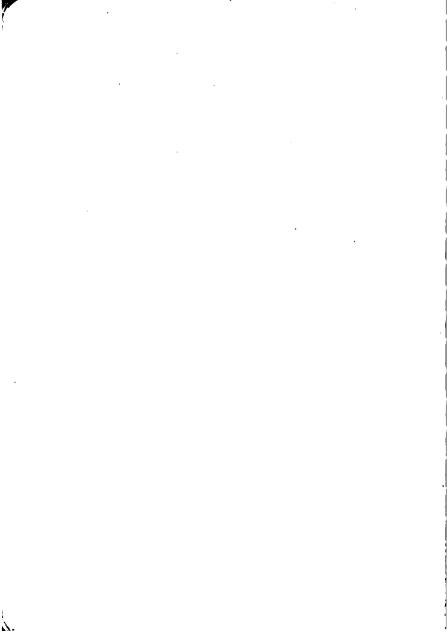
Nozzle Pressure Indi- cated by Pitot Gage.	Discharge, Gallons per Minute.	DRA STR TAI FIR LEN	PRESSURES REQUIRED AT HYDRANT OR FIRE ENGINE, WHILE STREAM IS FLOWING, TO MAINTAIN NOZZLE PRESSURES GIVEN IN FIRST COLUMN, THROUGH VARIOUS LENGTHS OF BEST QUALITY 3½-INCH RUBBER LINED HOSE.											
Nozzl cated	Disc I	200 Feet.	400 Feet.	600 Feet.	800 Feet.	1,000 Feet.	1,200 Feet.	1,500 Feet.	1,800 Feet.	Nozzle Pressure Indi- cated by Pitot Gage.				
20	350	31	41	50	60	70	80	94	109	20				
25	392	38	51	63	75	87	99	118	136	25				
80	429	46	60	75	89	103	118	139	161	80				
85	463	53	70	86	103	120	136	161	186	85				
40	496	61	79	98	117	136	155	183	211	40				
45	525	68	89	110	131	152	173	205	236	45				
50	554	76	99	122	145	168	192	226	261	50				
55	581	83	108	133	1 58	184	209	247	284	55				
60	607	90	117	144	172	199	226	267	308	60				
65	631	97	127	156	186	215	244	289		65				
70	655	105	136	167	199	230	262	309		70				
75	678	112	145	179	212	245	<b>27</b> 9	• • • •	• • • •	75				
80	700	119	155	191	226	262	297			80				
85	722	127	165	202	240	278	316		• • • •	85				
. 90	743	134	174	214	254	<b>2</b> 94				90				
95	763	141	183	225	267	309				95				
100	783	149	193	237	281		••••	••••	••••	100				

# 2-INCH SMOOTH NOZZLE - 3 1/2-INCH HOSE

Nozzle Pressure Indi- cated by Pitot Gage.	Discharge, Gallons per Minute.	DRA STR NOZ COI LEN	NT EAM ZZLE JUMN IGTH	PRES S OF UBBEI	Teet, Times and Three transfer three transfer tr	LE COLUMN	UAL:	WHAINT IN FINAL IN FI	RST OUS 31-	Nozzle Pressure Indi- cated by Pitot Gage.
20	532	33	44	55	65	76	87	109	130	20
25	594	41	54	67	80	93	106	133	159	25
80	651	49	64	80	96	111	127	158	189	80
35	703	57	75	93	111	129	147	183	219	85
40	752	65	85	105	126	146	166	207	247	40
45	797	72	95	118	140	163	185	231	276	45
50	841	80	105	130	155	180	205	255	305	50
55	881	88	116	143	170	197	225	279		55
60	920	96	126	155	185	214	244	303		60
65	958	104	136	168	200	232	263	<b> </b>		65
70	994	112	146	180	214	248	282		 	70
75	1,029	119	156	192	229	265	301	•••		75
80	1,063	127	166	205	243	282	••••			80
85	1,095	135	176	217	258	299		. <b></b>		85
90	1,128	143	186	229	272		٠.			90
95	1,158	151	196	241	286					95
100	1,189	158	206	253	301				•••	100

# AMMORLAŠ







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